



# **Finite Elements for Computational Fluid Dynamics**

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## **Basic equations**

#### **Conservation or transport equations**

- Conservation equations for:
  - Mass
  - Momentum
  - Energy
- Transport equations for:
  - Concentrations
  - Turbulence
  - etc.

System of non-linear, coupled, partial differential equations of convection-diffusion type

$$\frac{\partial \varphi}{\partial t} + U_j^* \frac{\partial \varphi}{\partial x_j} - \Gamma \frac{\partial^2 \varphi}{\partial x_j^2} - f = 0$$

Time derivative
 Convection term
 Diffusion term
 Source term





## **Basic equations**

Computational domain and boundary conditions

Boundary conditions:

Dirichlet type (fixed values) Neumann type (gradient type b.c.)



For unsteady problems also Initial Conditions are required

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## **Solution methods**

Usually used solution methods:

- Finite Difference Methods (FDM)
- Finite Volume Methods (FVM)

– Finite Element Methods (FEM)

Other Methods

- Spectral methods
- Lattice Boltzmann Methods
- Boundary Element Methods
- Etc.

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#### **Solution methods**



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FEM

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#### **Solution methods**









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# **Definition of describing equations**

Example: Navier-Stokes equations, steady-state, incompressible

Mass conservation:

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$$

Momentum conservation: (x-direction)

Momentum conservation: (y-direction)

$$u\frac{\partial u}{\partial x} + v\frac{\partial u}{\partial y} = -\frac{1}{\rho}\frac{\partial p}{\partial x} + v\left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2}\right)$$

$$u\frac{\partial v}{\partial x} + v\frac{\partial v}{\partial y} = -\frac{1}{\rho}\frac{\partial p}{\partial y} + v\left(\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2}\right)$$

- Coupled
- Non-linear
- 2. order for velocities —
- 1. order for pressure –





# **Definition of describing equations**

Navier-Stokes equations, 3D, unsteady, incompressible

$$\frac{\partial \mathbf{u}}{\partial \mathbf{x}} + \frac{\partial \mathbf{v}}{\partial \mathbf{y}} + \frac{\partial w}{\partial \mathbf{z}} = 0$$

*θ*Π.

$$\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z} = -\frac{1}{\rho} \frac{\partial p}{\partial x} + v \left( \frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} \right)$$
$$\frac{\partial v}{\partial t} + u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} + w \frac{\partial v}{\partial z} = -\frac{1}{\rho} \frac{\partial p}{\partial y} + v \left( \frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} + \frac{\partial^2 v}{\partial z^2} \right)$$
$$\frac{\partial w}{\partial t} + u \frac{\partial w}{\partial x} + v \frac{\partial w}{\partial y} + w \frac{\partial w}{\partial z} = -\frac{1}{\rho} \frac{\partial p}{\partial z} + v \left( \frac{\partial^2 w}{\partial x^2} + \frac{\partial^2 w}{\partial y^2} + \frac{\partial^2 w}{\partial z^2} \right)$$

In Index form

Mass conservation

Momentum conservation

$$\frac{\partial u_i}{\partial t_i} = 0$$
  
$$\frac{\partial u_i}{\partial t} + u_j \frac{\partial u_i}{\partial x_j} = -\frac{1}{\rho} \frac{\partial p}{\partial x_i} + \nu \frac{\partial^2 u_i}{\partial x_j^2}$$

Summation convention

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1 Definition of describin	ng equations
Mass conservation	$\frac{\partial u_i}{\partial x_i} = 0$ Example: Turbulent flow
Momentum conservation	$\frac{\partial \mathbf{u}_{i}}{\partial t} + \mathbf{u}_{j} \frac{\partial \mathbf{u}_{i}}{\partial \mathbf{x}_{j}} = -\frac{1}{\rho} \frac{\partial p}{\partial \mathbf{x}_{i}} + (\nu + \nu_{t}) \frac{\partial^{2} \mathbf{u}_{i}}{\partial \mathbf{x}_{j}^{2}}$
Turbulent viscosity	$v_t = c_\mu \frac{k^2}{\varepsilon}$
Turbulent kinetic energy	$\frac{\partial k}{\partial t} + u_i \frac{\partial k}{\partial x_i} = \frac{\partial}{\partial x_i} \left( \frac{v_t}{\sigma_{\varepsilon}} \frac{\partial k}{\partial x_i} \right) + v_t \left( \frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right) \frac{\partial u_i}{\partial x_j} - \varepsilon$
Dissipation rate	$\frac{\partial \varepsilon}{\partial t} + u_i \frac{\partial \varepsilon}{\partial x_i} = \frac{\partial}{\partial x_i} \left( \frac{v_t}{\sigma_{\varepsilon}} \frac{\partial \varepsilon}{\partial x_i} \right) + c_{1\varepsilon} \frac{\varepsilon}{k} v_t \left( \frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right) \frac{\partial u_i}{\partial x_j} - c_{2\varepsilon} \frac{\varepsilon^2}{k}$
Model constants	$c_{\mu} = 0.09;  \sigma_{k} = 1.0;  \sigma_{\epsilon} = 1.3;  c_{1\epsilon} = 1.44;  c_{2\epsilon} = 1.92$
Advection – diffusion equati	on $\frac{\partial \varphi}{\partial t} + u_j \frac{\partial \varphi}{\partial x_j} - \Gamma \frac{\partial^2 \varphi}{\partial x_j^2} - f = 0$

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# 2 Definition of computational domain and boundary conditions

## Example backward-facing step









# 2 Definition of computational domain and boundary conditions















# **3** Dividing of the domain into Finite Elements



## Various types of elements









**Dividing of the domain into Finite Elements** 





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#### **Elements**











**Elements** 

Simple grids



		5	10	18	24	29	34	39	44	50	58	63	68	73
		4	9	17	23	28	33	38	43	49	57	62	67	72
		3	8	16	22	27	32	37	42	48	56	61	66	71
		2	7	15	21	$\sum_{i}$	31	36	X	47	55	60	65	70
-		1	6	14	777,	20 2 19 3 12 1	- 5 30	35	746 52 5:	ŗIJ	54	59	64	69
·		<u> </u>			/			5.					Source: Ta	aylor & Hughes
	Curvil	inea	r elen	ne	nts									







# **Insertion: Characteristic grid structures**

# **Cartesian Grid**



All grid lines are straight lines parallel to the coordinate axis All cells are rectangular All angles are right angles

Partial differentials can be build in 1D

For FDM

```
Example program
delx = Lx/(nx-1)
dely = Ly/(ny-1)
delz = Lz/(nz-1)
 do i = 1.nx
 do j = 1,nz
 do k = 1,nz
   x (i,j,k) = delx^{*}(i-1)
   y(i,j,k) = delx^{*}(j-1)
   z(i,j,k) = delz^{*}(k-1)
   u(i,j,k) = 0.0
  end do
 end do
  end do
```













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#### **Characteristic grid structures**





# Structured grids

2D
Two regimes of grid lines
Cells with four corners
Each grid point has 4 neighbors

Body-fitted coordinates by using curvilinear grid lines

**Unstructured grids** 

No regimes of grid lines Cells can have different shapes e.g. triangles ... No of neighbors are different 3D three Hexahedron six

Prisms...





## **Bock-structured grids**



# Turbine blades (Geometry is periodic)

Each block consists of a structured grid

On the block boundaries the grid points are identical between neighbor blocks

Grid lines should be smooth on block boundaries

Advantages:

Suitable for complex geometries Easy to parallelize



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#### **Unstructured grids**

Unlike for structured meshes, where band matrices occur, for unstructured meshes, unstructured, sparse matrices arise instead.



Structure of the matrices is symmetric

Matrices are not stored completely, instead only the relevant parts are stored, for this special storage technologies exists.









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In each element an approximation for the required quantity  $\phi$  is defined. This means it is defined, how the quantity behaves within the element.

Examples:

2D linear:

$$\widehat{\phi}(\mathbf{x},\mathbf{y}) = \alpha_1 + \alpha_2 \mathbf{x} + \alpha_3 \mathbf{y}$$

2D quadratic:  $\widehat{\phi}(\mathbf{x}, \mathbf{y}) = \alpha_1 + \alpha_2 \mathbf{x} + \alpha_3 \mathbf{y} + \alpha_4 \mathbf{x}^2 + \alpha_5 \mathbf{x} \mathbf{y} + \alpha_6 \mathbf{y}^2$ 

The coefficients are the solution







## **Definition of approximation**

Examples:

$$\widehat{\Phi} = \alpha_0 + \alpha_1 \mathbf{X}$$
$$\widehat{\Phi} = \alpha_0 + \alpha_1 \mathbf{X} + \alpha_2 \mathbf{X}^2$$

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$$\widehat{\Phi} = \alpha_0 + \alpha_1 \mathbf{X} + \alpha_2 \mathbf{Y}$$

$$\widehat{\Phi} = \alpha_0 + \alpha_1 \mathbf{X} + \alpha_2 \mathbf{Y} + \alpha_3 \mathbf{X} \mathbf{Y}$$

$$\widehat{\Phi} = \alpha_0 + \alpha_1 \mathbf{X} + \alpha_2 \mathbf{y} + \alpha_3 \mathbf{x}^2 + \alpha_4 \mathbf{y}^2 + \alpha_5 \mathbf{x} \mathbf{y}$$





The approximations must fulfill certain continuity conditions:

1.) Conforming elements

The approximation on neighbor element boundaries must be identical

These are called *conforming elements*.

Only conforming elements are considered here.

EXAMPLE: Non-conforming velocity approximation



Because the approximations are different in each element, this acts as a source or sink between the elements







# 2.) $C_{n-1}$ continuity requirement

If the integrant contains derivatives of n-th order, the approximations have to be continuous up to derivatives of (n-1)-th order.

For the Navier-Stokes equations this means:

$$\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} = -\frac{1}{\rho} \frac{\partial p}{\partial x} + v \left( \frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right)$$

Velocities are 2nd order, pressure 1st order

- $\Rightarrow$  Velocity approximation must be continuous and differentiable
- $\Rightarrow$  Pressure approximation must be continuous

(The continuity requirements can be reduced by applying Galerkin method by means of Green-Gauss theorem, see later)

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To fulfill the  $C_{n-1}$  continuity requirements the polynomial approximations are not suitable. Between the coefficients of neighboring elements constraints has to be satisfied.

It is much easier to use node values instead of coefficients to prescribe the approximation



The continuity can be satisfied easier when expressing the approximation by node values

$$\widehat{\phi} = \widehat{\phi}^1 + \left(\widehat{\phi}^2 - \widehat{\phi}^1\right) \frac{(\mathbf{x} - \mathbf{x}_1)}{\mathbf{x}_2 - \mathbf{x}_1}$$





Generalized: 
$$\widehat{\phi}(x_i) = \sum \widehat{\phi}^k N^k(x_i) < k = 1,... \text{ No nodes } > 1$$

Linear Approximation:  $\widehat{\phi}^{(e)} = \widehat{\phi}^{1(e)} N^{1(e)} + \phi^{2(e)} N^{2(e)}$ 

N are the shape or trial functions (der index (e) means that it is related to the local element numbering).

Introducing the dimensionless coordinate  $\xi = \frac{x - x_1}{x_2 - x_1}$ 

the shape functions are

 $N^{1(e)} = 1 - \xi \quad \text{and} \qquad N^{2(e)} = \xi$ 

The shape function characterize the influence of a node value to the solution.

The shape function must be 1 in the corresponding node and must be 0 in all other nodes



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By a global numbering of all nodes the local approximations can be put together to a global approximation.

$$\widehat{\phi}(\mathbf{x}_i) = \sum \widehat{\phi}^k \mathbf{N}^k(\mathbf{x}_i) \quad < k = 1,..., \text{ Number of nodes } > 1$$

The global shape functions are composition of the local shape functions

Example: 1D linear









**Definition of approximation** 

$$\widehat{\varphi}(x_i) = \sum \widehat{\varphi}^k N^k(x_i) < k = 1,... \text{ no nodes } >$$









# **Definition of approximation**

For 2D linear triangles the following shape function for node k is obtained









The introduction of global shape functions simplifies the formal description,

For programming only the element shape functions are usually used.

This is explained at a 1D example.



elements with local numbering

Elements with global numbering

Applied is a linear approximation





## **Definition of approximation**

Approximation for element 1: Approximation for element 2:

$$\phi^{(e)} = \alpha_1 + \alpha_2 x$$
$$\phi^{(e)} = \beta_1 + \beta_2 x$$

As already mentioned with these approximations it is difficult to satisfy the continuity requirements. There are constrains between the coefficients. Therefore the approximations are expressed in node values

$$\widehat{\phi}^{(e)} = \widehat{\phi}^{1(e)} + \left(\phi^{2(e)} - \phi^{1(e)}\right) \frac{x - x^{1(e)}}{x^{2(e)} - x^{1(e)}}$$

This results in

for element

for element 1:  

$$\widehat{\phi}^{(e)} = \widehat{\phi}^1 + \left(\widehat{\phi}^2 - \widehat{\phi}^1\right) \frac{x - x^1}{x^2 - x^1}$$
for element 2:  

$$\widehat{\phi}^{(e)} = \widehat{\phi}^2 + \left(\widehat{\phi}^3 - \widehat{\phi}^2\right) \frac{x - x^2}{x^3 - x^2}$$

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### **Definition of approximation**

So the continuity is satisfied without additional measures.

By introducing a dimensionless coordinate for each element:

$$\xi = \frac{x - x^{1(e)}}{x^{2(e)} - x^{1(e)}}$$

the approximation can be written in the form

$$\widehat{\phi}^{(e)} = \widehat{\phi}^{1(e)} \left(1 - \xi\right) + \phi^{2(e)} \xi$$

or with the shape functions

$$\widehat{\phi}^{(e)} = \widehat{\phi}^{1(e)} \mathbf{N}^{1(e)} + \phi^{2(e)} \mathbf{N}^{2(e)}$$

Where the shape functions are defined as

$$N^{1(e)} = 1 - \xi$$
 and  $N^{2(e)} = \xi$ 

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#### **Definition of approximation**

The continuity is automatically satisfied by using node values



But differentiability is not yet obtained

This could be obtained by introducing also the derivatives on the nodes as generalized node values



This works, but it is very complicated in 2D and 3D



Example:



If not Galerkin method is applied (reduction of continuity requirements!) for higher order differential equations mostly the higher order differential equation is transformed into a system of differential equation of first order.

Basic equation  $\frac{\partial^2 u}{\partial x^2} + f = 0$ 

Introduction of function g  $g = \frac{\partial u}{\partial x}$ Resulting system of 1<sup>st</sup> order equation  $\frac{\partial g}{\partial x} + f =$ 

$$\frac{\partial g}{\partial x} + f = 0$$
$$\frac{\partial u}{\partial x} - g = 0$$

By introducing the vorticity vector the Navier-Stokes equations can be transferred into a system of 6 equations of first order

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In opposition to the Finite Difference Method, where a direct discretization of the differential equations is undertaken, the FEM is based on a variation principle:

$$\Pi = \iiint_{\Omega} F\left(\phi, \frac{\partial \phi}{\partial x}, \frac{\partial \phi}{\partial y}, \frac{\partial \phi}{\partial z}, \frac{\partial^2 \phi}{\partial x^2}, \dots, x, y, z\right) dx dy dz = \min$$

 $\phi$  is the solution function looked for. It can be a single function or a set of unknown functions (e.g. for the Navier-Stokes equations  $\phi$  stands for the velocity components and the pressure).

In structural mechanics an equivalent variation principle to the differential equation formulation exists (e. g. principle of virtual work).

For the Navier-Stokes equations an equivalent variation principle is not known! As a consequence an approximation must be used.





## Method of weighted residuals (MWR)

Partial Differential equation (PDE)

$$D(\Phi) = f$$
 im Gebiet  $\Omega$ 

Approximation of the solution

$$\hat{\Phi} = \sum_{k=1}^{n} \alpha^k \cdot \phi^k$$

 $\phi^k$ ...linear independent functions  $\alpha^k$ ...coefficients (looked for)

Introduced into the in PDE results in a residuum  $\varepsilon = D(\hat{\Phi}) - f$ 

Postulated:

$$\int w^{k} \cdot \varepsilon d\Omega = \int_{\Omega} w^{k} \cdot (D(\hat{\Phi}) - f) d\Omega = 0 < k = 1, \dots, n > 0$$

linear independent weighting functions

There are several methods to choose the weighting functions





#### **Discrete form of the describing equations**

#### **Least Square Method**

```
For the LSM it is postulated
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$$\iiint_{\Omega} \varepsilon^2 d\Omega = \min$$

This is equivalent to

$$\iiint_{\Omega} \frac{\partial \epsilon}{\partial \alpha^k} \epsilon \ d\Omega = 0$$

Compared to the general MWR approach this results in the following weighting functions

$$\mathbf{w}^{k} = \frac{\partial \boldsymbol{\varepsilon}}{\partial \boldsymbol{\alpha}^{k}}$$

<u>Note</u>: Using LSM no reduction of the continuity requirements can be obtained by partial integration (Green-Gauss theorem)

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FEM

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#### **Galerkin Formulation**

This is the mostly used formulation for the Navier-Stokes equation.

The weighting function are chosen equal to the shape functions

$$w^k = N^k$$

This results in a variation principle

$$\iiint_{\Omega} N^k \, \epsilon \, d\Omega = 0$$







#### **Collocation method**

Using the collocation method Dirac-functions are used as weighting functions

$$\mathbf{w}^{k} = \delta \left( \mathbf{x}_{i} - \mathbf{x}_{i}^{k} \right) \qquad \delta \left( \mathbf{x}_{i} - \mathbf{x}_{i}^{k} \right) = \begin{cases} 0 & \text{for } \left( \mathbf{x}_{i} \neq \mathbf{x}_{i}^{k} \right) \\ \infty & \text{for } \left( \mathbf{x}_{i} = \mathbf{x}_{i}^{k} \right) \end{cases} \qquad \int_{-\infty}^{\infty} \delta \, d\mathbf{x} = 1$$

This is equivalent to the postulation, that the residuals at given points (collocation points) is zero.

$$\iiint_{\Omega} \epsilon \cdot \delta \left( x_{i}^{} - x_{i}^{k} \right) d\Omega = \epsilon \big|_{x_{i}^{k}} = 0$$







#### **Subdomain Method**

The weighting function is chosen to be 1 in the subdomain and 0 outside the subdomain

This means the residuum zero in an "integral" way in the subdomain.

The weighting function is given as

$$\mathbf{w}^{k} = \begin{cases} 1 & \text{in } \mathbf{D}_{m} \\ 0 & \text{out of } \mathbf{D}_{m} \end{cases}$$

The subdomain method is equivalent to the Finite Volume Method (FVM)

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## Assembling of the relevant matrices

Calculation of the element matrices

Based on the method of weighted residuals

$$\iiint_{\Omega} w^{k} \varepsilon d\Omega = 0 \quad < k = 1, ..., no \text{ of nodes} >$$

the integral can be expressed by the sum of the integrals over all elements

$$\iiint_{\Omega} w^{k} \varepsilon d\Omega = \sum \iiint_{\Omega^{(e)}} w^{k(e)} \varepsilon d\Omega^{(e)}$$

This results in a linear system of equations (LSE)

# <u>A</u>∮=b

The matrix A can be assembled from the element matrices, also the vector of the right-hand side is calculated from all element contributions

Vector  $\phi$  are the node values, searched for.





Assembling of the relevant matrices

$$\underline{A} = \sum \underline{A}^{(e)}$$
 und  $\underline{b} = \sum \underline{b}^{(e)}$ 

The element matrices depend on the equations, the types of elements, approximations etc.

For linear PDEs and rather simple types of elements the element matrices can be calculated directly. For non-linear PDEs and/or more complex elements the element matrices must be calculated numerically.

The calculation of the element matrices will be shown later at an example.





#### Assembling of the relevant matrices



FEM





#### Introduction of boundary conditions

Through the assembly of the element matrices one obtains the global linear system of equations

Dirichlet boundary conditions must be introduces into this LSE

Neuman boundary conditions result in an additional contribution to the global matrix

$$\begin{bmatrix} a_{11} & a_{12} & a_{13} & a_{14} \\ a_{21} & a_{22} & a_{23} & a_{24} \\ a_{31} & a_{32} & a_{33} & a_{34} \\ a_{41} & a_{42} & a_{43} & a_{44} \end{bmatrix} \cdot \begin{bmatrix} \widehat{\phi}_1 \\ \widehat{\phi}_2 \\ \widehat{\phi}_3 \\ \widehat{\phi}_4 \end{bmatrix} = \begin{bmatrix} b_1 \\ b_2 \\ b_3 \\ b_4 \end{bmatrix}$$

Example: Diriclet b.c.

Node value 1 is given

 $\widehat{\phi}_1 = \phi_{\text{fix}}$ 

$$\begin{bmatrix} 1 & 0 & 0 & 0 \\ a_{21} & a_{22} & a_{23} & a_{24} \\ a_{31} & a_{32} & a_{33} & a_{34} \\ a_{41} & a_{42} & a_{43} & a_{44} \end{bmatrix} \cdot \begin{bmatrix} \hat{\phi}_1 \\ \hat{\phi}_2 \\ \hat{\phi}_3 \\ \hat{\phi}_4 \end{bmatrix} = \begin{bmatrix} \phi_{\text{fix}} \\ b_2 \\ b_3 \\ \hat{\phi}_4 \end{bmatrix}$$









FEM





#### Linearization

**Basic equations** 



Stokes-Linearization:

Picard-Iteration:

Convection term is completely taken from the previous iteration step

$$u\frac{\partial u}{\partial x} \approx u^{old}\frac{\partial u^{new}}{\partial x}$$

Robust, slower convergence, large convergence radius

Newton-Iteration:

fast convergence, small convergence radius







#### **Steady-state convection-diffusion equation**

**Basic equation** 

Approximation within the element







$$\varepsilon = \mathbf{U}_{j}^{*} \frac{\partial \mathbf{\phi}}{\partial \mathbf{x}_{j}} - \Gamma \frac{\partial^{2} \mathbf{\phi}}{\partial \mathbf{x}_{j}^{2}} - \mathbf{f}$$

residuals

$$\iiint_{\Omega} w^{k} \varepsilon \, d\Omega = \sum_{\text{Elemente}} w^{k(e)} \varepsilon \, d\Omega$$

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#### Steady-state convection-diffusion equation

Galerkin formulation

$$\mathbf{w}^{\mathbf{k}} = \mathbf{N}^{\mathbf{k}}$$

$$\iint_{\Omega} N^{k} \epsilon \, d\Omega = \sum_{\text{elements}} \iint_{El \, i} N^{k(e)} \epsilon \, d\Omega = 0$$

with

$$\varepsilon = \left( U_{j}^{*} \frac{\partial N^{l}}{\partial x_{j}} - \Gamma \frac{\partial^{2} N^{l}}{\partial x_{j}^{2}} \right) \dot{\phi}^{l} - f$$

$$\sum_{\text{Elemente}} \iiint_{\text{El i}} N^{k} \Biggl( \Biggl( U_{j}^{*} \frac{\partial N^{l}}{\partial x_{j}} - \Gamma \frac{\partial^{2} N^{l}}{\partial x_{j}^{2}} \Biggr) \overset{\land}{\phi}^{l} - f \Biggr) d\Omega = 0$$













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#### **Unsteady convection-diffusion equation**

Basic equation

Approximation within the elements

Residuum

MWR





Galerkin formulation

$$\mathbf{w}^{k} = \mathbf{N}^{k}$$

FEM





#### **Unsteady convection-diffusion equation**

**Discretized equations** 









=> In FEM usually an implicit time descretization is applied





#### **Example: 1D Helmholtz equation**

$$\frac{\partial^2 \phi}{\partial x^2} - \lambda^2 \phi = 0 \qquad \lambda = 0.5$$

Boundary conditions  $\phi(0)=1; \quad \phi(2)=1.54308$ 

Exact solution:

$$\phi = c_1 e^{-\lambda x} + c_2 e^{\lambda x}$$
 with  $c_1 = c_2 = 0.5$ 

#### MWR

$$\int_{0}^{2} w^{k} \left( \frac{\partial^{2} \widehat{\phi}}{\partial x^{2}} - \lambda^{2} \widehat{\phi} \right) dx = 0$$

Approximation

$$\widehat{\boldsymbol{\varphi}} = \widehat{\boldsymbol{\varphi}}^k \mathbf{N}^k$$

In the direct form the approximation must be continuous and differentiable.

The requirement can be reduced by applying Galerkin method and the Green Gauss theorem







#### **Example: 1D Helmholtz equation**

Galerkin formulation

$$\mathbf{w}^{k} = \mathbf{N}^{k}$$

Weighting functions = Shape functions

C<sub>n-1</sub>- continuity

Integral contains derivations 2. order => approximation continuous and differentiable

k and I count from 1 to the no of nodes n

By applying Green-Gauss theorem the continuity requirements can be reduced by one order.





# Green-Gauss theorem









**Example: 1D Helmholtz equation** 

Weak formulation

$$\int_{0}^{2} \left( \frac{\partial \mathbf{N}^{k}}{\partial x} \frac{\partial \mathbf{N}^{1}}{\partial x} + \lambda^{2} \mathbf{N}^{k} \mathbf{N}^{1} \right) dx \left[ \widehat{\phi}^{1} = \underline{A} \widehat{\phi} = 0 \right]$$
(Change of sign!!)

The resulting surface integral can be neglected since Diriclet boundary conditions are given. For Neuman boundary conditions the surface integral has to be calculated and results in an addition to the system matrix.

In the weak formulation the integral contains only derivatives of first order. Therefor only  $C_0$ -continuity is required. (This is the big advantage of the Galerkin formulation)

Solution with 4 linear elements







#### **Example: 1D Helmholtz equation**

Calculation of the global matrix from the local element matrices

$$\begin{bmatrix} \sum_{x_{i}}^{x_{i+1}} \left( \frac{\partial N^{k}}{\partial x} \frac{\partial N^{1}}{\partial x} + \lambda^{2} N^{k} N^{1} \right) dx \end{bmatrix} \widehat{\phi}^{1} = \sum_{x_{i}} \underline{A}^{(e)} \widehat{\phi} = 0$$
Element matrix
Dimensionless coordinate
$$\xi = \frac{x - x_{i}}{L} \quad \text{with} \quad L = x_{i+1} - x_{i}$$

$$N^{i} = N^{i+1}$$

Linear shape functions

$$N^{i} = 1 - \xi;$$
  $N^{i+1} = \xi$ 

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X<sup>i</sup>

 $\mathbf{X}^{i+1}$ 

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FEM





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#### **Example: 1D Helmholtz equation**

with

$$dx = L d\xi$$

and

$$\frac{\partial \mathbf{N}^{k}}{\partial \mathbf{x}} = \frac{\partial \mathbf{N}}{\partial \xi} \frac{\partial \xi}{\partial \mathbf{x}} = \frac{1}{\mathbf{L}} \frac{\partial \mathbf{N}}{\partial \xi}$$

Tis results in

$$\underline{\underline{A}}^{(e)} = \int_{0}^{1} \left( \frac{1}{L^{2}} \frac{\partial N^{k}}{\partial \xi} \frac{\partial N^{1}}{\partial \xi} + \lambda^{2} N^{k} N^{1} \right) \cdot L d\xi$$

In components

$$\underline{\underline{A}}^{(e)} = \int_{0}^{1} \begin{bmatrix} \frac{1}{L} \left( \frac{\partial N^{1}}{\partial \xi} \right)^{2} + \lambda^{2} L \left( N^{1} \right)^{2} & \frac{1}{L} \left( \frac{\partial N^{1}}{\partial \xi} \frac{\partial N^{2}}{\partial \xi} \right) + \lambda^{2} L \left( N^{1} N^{2} \right) \\ \frac{1}{L} \left( \frac{\partial N^{1}}{\partial \xi} \frac{\partial N^{2}}{\partial \xi} \right) + \lambda^{2} L \left( N^{1} N^{2} \right) & \frac{1}{L} \left( \frac{\partial N^{2}}{\partial \xi} \right)^{2} + \lambda^{2} L \left( N^{2} \right)^{2} \end{bmatrix} d\xi$$

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#### **Example: 1D Helmholtz equation**

with 
$$\frac{\partial N^1}{\partial \xi} = -1$$
 and  $\frac{\partial N^2}{\partial \xi} = 1$   
and  $\int_0^1 \xi^2 d\xi = \frac{1}{3};$   $\int_0^1 (1-\xi)\xi d\xi = \frac{1}{6}$  and  $\int_0^1 (1-\xi)^2 d\xi = \frac{1}{3}$ 

With L = 0.5 and  $\lambda$  = 0.5 this results in the element matrix

$$\underline{\mathbf{A}}^{(e)} = \begin{bmatrix} \frac{1}{L} + \lambda^2 \frac{L}{3} & -\frac{1}{L} + \lambda^2 \frac{L}{6} \\ -\frac{1}{L} + \lambda^2 \frac{L}{6} & \frac{1}{L} + \lambda^2 \frac{L}{3} \end{bmatrix} = \begin{bmatrix} 2.042 & -1.979 \\ -1.979 & 2.042 \end{bmatrix}$$





#### **Example: 1D Helmholtz equation**











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#### **Example: 1D Helmholtz equation**

#### Solution using two quadratic elements



Shape functions 
$$\begin{split} N^{1(e)} &= (1-\xi)(1-2\xi) \\ N^{2(e)} &= 4\xi(1-\xi) \\ N^{3(e)} &= -\xi(1-2\xi) \end{split}$$

In analogy to the linear elements the element matrices can be calculated





#### **Example: 1D Helmholtz equation**

**Element matrix** 

$$\underline{\mathbf{A}}^{(e)} = \begin{bmatrix} \frac{7}{3L} + \frac{4L}{30} & \frac{-8}{3L} + \frac{2L}{30} & \frac{1}{3L} - \frac{L}{30} \\ \frac{-8}{3L} + \frac{2L}{30} & \frac{16}{3L} + \frac{16L}{30} & \frac{-8}{3L} + \frac{2L}{30} \\ \frac{1}{3L} - \frac{L}{30} & \frac{-8}{3L} + \frac{2L}{30} & \frac{7}{3L} + \frac{4L}{30} \end{bmatrix}$$

Both element matrices will be summed up and the boundary conditions must be introduced.





#### **Example: 1D Helmholtz equation** Global matrix 8 3L<sub>1</sub> $\frac{1}{3L_1}$ $\overline{3L_1}$ $\frac{8}{3L_1}$ <u>16</u> <u>8</u> 3L $3L_1$ $\frac{8}{3L_1}$ $\frac{1}{3L_1}$ $\frac{8}{3L_2}$ $\frac{1}{3L_2}$ 3L1+ i3L<sub>2</sub> $\frac{8}{3L_2}$ 8 <u>16</u> 3L<sub>2</sub> $\overline{3L_2}$ $\frac{8}{3L_2}$ $\frac{7}{3L_2}$ $3L_2$



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#### **Gauss Integration**

For linear PDEs and rather simple types of elements the element matrices can be calculated directly. For non-linear PDEs and/or more complex elements the element matrices must be calculated numerically.





	X	Y	ω
1	0.2113	0.2113	0.25
2	0.7 <i>8</i> 87	0.2113	0.25
3	0.2113	o.7 <i>88</i> 7	0.25
4	0.7 <i>8</i> 87	<i>0.78</i> 87	<i>6.25</i>







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#### **Navier-Stokes equation**

Mass conservation

$$\frac{\partial \mathbf{u}_{i}}{\partial \mathbf{x}_{i}} = 0$$

 $\partial \mathbf{u}$ 

Momentum conservation

Approximations

$$u_i = N^k u_i^k$$

 $N^{k}$  and  $M^{k}$  Shape functions  $\hat{u}_{i}^{k}$  Node values

$$\hat{\mathbf{p}} = \mathbf{M}^k \hat{\mathbf{p}^k}$$





#### **Navier-Stokes equation**

#### Momentum equations

$$\begin{split} &\int_{\Omega} \left[ N^{k} N^{l} U_{i}^{l} + \widetilde{U}_{j} N^{k} \frac{\partial N^{l}}{\partial x_{j}} U_{i}^{l} + \upsilon_{eff} \frac{\partial N^{k}}{\partial x_{j}} \left( \frac{\partial N^{l}}{\partial x_{j}} U_{i}^{l} + \frac{\partial N^{l}}{\partial x_{i}} U_{j}^{l} \right) \right] d\Omega \\ &- \int_{\Gamma} \upsilon_{eff} N^{k} \left( \frac{\partial N^{l}}{\partial x_{j}} U_{i}^{l} + \frac{\partial N^{l}}{\partial x_{i}} U_{j}^{l} \right) n_{j} d\Gamma - \int_{\Omega} \frac{1}{\rho} \frac{\partial N^{k}}{\partial x_{i}} M^{q} P^{q} + \int_{\Gamma} N^{k} M^{q} P^{q} n_{i} d\Gamma = 0 \end{split}$$

$$< k \_ 1,...,n >, < l = 1,...,n >, < q = 1,...,m >$$

Continuity equation

$$\int_{\Omega} M^{q} \frac{\partial N^{l}}{\partial x_{i}} U_{i}^{l} d\Omega = 0 \qquad \langle q = 1, ..., m \rangle, \quad \langle l = 1, ..., n \rangle$$





#### **Navier-Stokes equation**

- Minimum continuity requirements:
- Continuous approximation for velocity components
- Discontinuous approximatio for pressure



Tri-linear Hexahedron for velocity components



Constant pressure







#### **LBB** Condition

### Ladyzhenskaya-Babuška-Brezzi condition

The approximations for velocity and pressure cannot be defined independently.

For the Navier-Stokes equation the velocity components can be interpreted as main variables (degrees of freedom),

The pressure can be seen as constrains

It is necessary to have more degrees of freedom as constrains

To guaranty a unique solution, the approximations must fulfill the LBB condition, this condition is very complicated and cannot be discussed here.

As a rule of thumb, it can be said, that the approximation of the velocities should be one order higher compared to the approximation of the pressure

$$\sup_{v\in V, v
eq 0} rac{b(v,q)}{\|v\|_V} \geq eta \|q\|_Q$$





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#### **LBB Condition**

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Example of elements, satisfying the LBB condition



○ Velocity node▽ Pressure node





#### **LBB Condition**

Elements failing the LBB condition, but still performing reasonable

Velocity node

Pressure node



Problem with decoupling of different pressure modes

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#### **LBB Condition**

Problem with decoupling of different pressure modes

Checkerboard oscillation



Cure: Smoothing of the pressure





#### **Convection-diffusion problem**









#### **Upwind discretization**

"Streamline-upwind" Petrov-Galerkin Method

Approximation:  $\hat{\phi} = N^k \Phi^k$  $\mathbf{\tilde{N}}^k$ **Galerkin**:  $\int N^k \varepsilon d\Omega = 0$  $\mathbf{N}^{\mathbf{k}}$ Petrov-Galerkin:  $\int_{\Omega} N^k \varepsilon d\Omega = 0$ Residuum:  $\varepsilon = \frac{\partial \hat{\phi}}{\partial t} + U_j^* \frac{\partial \hat{\phi}}{\partial x_i} - \Gamma \frac{\partial^2 \hat{\phi}}{\partial x_i^2} - f$ 





**Upwind discretization** 

# SUPG weighting function

$$\widetilde{N}^{k} = \begin{cases} N^{k} + \frac{\partial N^{k}}{\partial x_{j}} \frac{U_{j}}{\sqrt{U_{i}U_{i}}} \cdot k'' & \text{für } Re^{(e)} > 2\\ N^{k} & \text{für } Re^{(e)} \le 2 \end{cases}$$

$$Re^{(e)} = \frac{U^{(e)}h}{\Gamma}$$
$$k'' = \left(\operatorname{coth}(Re^{(e)}/2) - 2/Re^{(e)}\right) \cdot h/2$$



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#### **Example ship propeller**

## Tip vortex



Prediction of vortex cavitation









#### **Example ship propeller**







#### **Example ship propeller**







#### <u>Summary</u>

- Finite Element Method is very flexible
  - Different element types (tetraether, hexaeder, curvilinear elements, .....)
  - Different approximation (linear, quadratic .....)
- Unstructured grids
- Mostly used: Galerkin method
  - Green-Gauss theorem
  - Reduction of continuity requirements
- Streamline upwind Petrov Galerkin method for convection dominated flows
  - Skew-symmetric weighting function according to flow direction

Error estimation possible

Adaptive grid refinement possible

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